

1. Ans.(a)
2. Ans.(b)
3. Ans.(d)
4. Ans.(a)
5. Ans.(d)
6. Ans.(b)
7. Ans.(a)
8. Ans. (b)

$$C_p = 1005 \text{ J/kg} \cdot \text{C} = \frac{R\gamma}{\gamma-1} = \frac{287 \times \gamma}{(\gamma-1)}$$

So $\gamma = 1.399 \approx 1.4$

$$T_2 = T_1 + \frac{V^2}{2C_p}$$

$$T_2 = 358.95 \text{ K}$$

$$p_2 = p_1 \left(\frac{T_2}{T_1} \right)^{\frac{1.4}{0.4}} = 10 \times \left(\frac{358.95}{298} \right)^{3.5} = 192 \text{ KPa}$$

9. Ans.(d)
10. Ans.(a)
11. Ans.(c)

Since the deflection in both the ropes is same and the lengths are unequal, then

$$\frac{P_1}{P_2} = \frac{L_2}{L_1} = \frac{6}{8} = \frac{3}{4}$$

12. Ans.(b)

As hinge does not take any moment, therefore reaction at B will be (R_B)

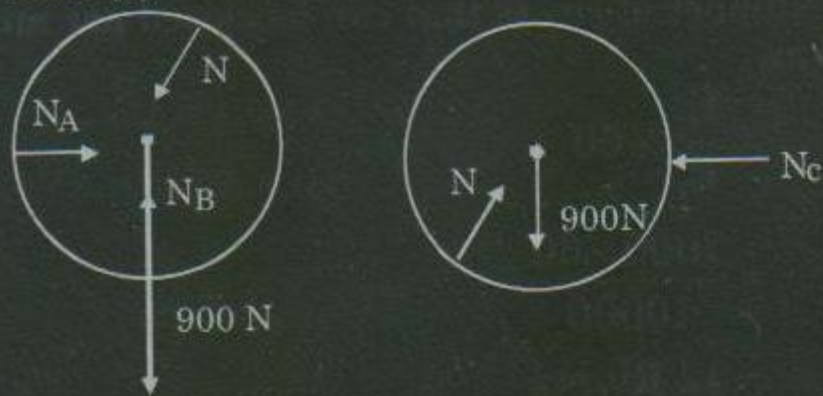
$$4W = 5R_E$$

$$10R_E + 3R_B - 9W = 0$$

$$R_B = \frac{W}{3}$$

13. Ans.(d)

14. Ans.(c)



$$\frac{N}{\sqrt{2}} = 900$$

$$N_c = \frac{N}{\sqrt{2}} = 900N$$

$$N_A = \frac{N}{\sqrt{2}} = 900N, N_B = 900 + \frac{N}{\sqrt{2}} = 1800N$$

15. Ans.(c)

16. Ans.(d)

17. Ans.(d)

18. Ans.(c)

19. Ans.(b)

$$\text{Time} = \frac{20 \times 60}{.2 \times 300} = 20 \text{sec}$$

20. Ans.(a)

Let B.E.Q be 'Q' then

Total cost = 80000 + 10Q

Revenue = 30Q

$$30Q = 80000 + 10Q$$

$$20Q = 80000$$

$$Q = 4000$$

21. Ans. (b)

22. Ans.(b)

$$F(x) = 3.46 \sin x + 2 \cos x = 4 \sin(x + \frac{\pi}{6})$$

Has maximum at $x = \frac{\pi}{6}$

23. Ans.(d)

24. Ans.(c)

$$\frac{dz}{d\omega} = \frac{\partial z}{\partial u} = i e^{-v} (\cos u + i \sin u) = iz$$

$$\text{So } \frac{d\omega}{dz} = \frac{1}{iz}$$

So ω ceases to be analytic for $z=0$

25. Ans.(c)

$$\frac{x^2}{a^2} + \frac{y^2}{b^2} = 1$$

$$\frac{x}{a^2} + \frac{y}{b^2} \frac{dy}{dx} = 0$$

$$\frac{1}{a^2} + \frac{y}{b^2} \frac{d^2y}{dx^2} + \frac{\left(\frac{dy}{dx}\right)^2}{b^2} = 0$$

Eliminating a and b from the equations we get

$$xy \frac{d^2y}{dx^2} + x \left(\frac{dy}{dx}\right)^2 - y \left(\frac{dy}{dx}\right) = 0$$

26. Ans.(b)

Solution: $\sum \lambda_i = \text{Trace}(A)$

$$\lambda_1 + \lambda_2 + \lambda_3 = \text{trace}(A)$$

$$= 2 + (-1) + 0 = 1$$

$$\lambda_1 = 3$$

Now

$$\therefore 3 + \lambda_2 + \lambda_3 = 1$$

$$= \lambda_2 + \lambda_3 = -2$$

Only choice (b) satisfies this condition

27. Ans.(b)

Solution: $\frac{dy}{dx} = \frac{y}{x} \left[\log\left(\frac{y}{x}\right) + 1 \right]$, putting $v = \frac{y}{x}$

$$\text{We get } \frac{dy}{dx} = v + x \frac{dv}{dx}$$

$$\frac{dx}{x} = \frac{dv}{v \log v}$$

$$\text{Log}x + \text{log}c = \text{log}(\text{log}v)$$

$$v = e^{cx}$$

$$Y = xe^{xc}$$

28. Ans.(d)

Solution: Both $f(x)$ and $f(-x)$ changes their sign once. Hence there are only one real root on both positive and negative sides on x-axis.

29. Ans.(c)

Exp. Maximum surface roughness

$$R_a = \frac{f^2}{18\sqrt{3}R}$$

$$f^2 = 18\sqrt{3} \times R \times R_a$$

$$= 18\sqrt{3} \times 1.8 \times 3 \times 10^{-3}$$

$$f^2 = 0.168 \text{ mm / rev}$$

$$f = 0.410 \text{ mm / rev}$$

30. Ans.(d)

Exp. $\therefore \sigma = K\epsilon^n$

$$\ln \sigma = \ln k + n \ln \epsilon$$

$$n = \frac{\ln \sigma - \ln k}{\ln \epsilon} = \frac{\ln 515 - \ln 2015}{\ln 0.1} = 0.592$$

$$n \approx 0.60$$

31. Ans.(a)

Exp. Cauchy's integral theorem is

$$F(a) = \frac{1}{2\pi i} \int_c \frac{f(z)}{z-a} dz$$

$$\text{i.e. } \int_c \frac{f(z)}{z-a} dz = 2\pi i f(a)$$

$$\text{Now } \int_c \frac{z^3 - 6}{3z - i} dz = \frac{1}{3} \int_c \frac{z^3 - 6}{\left(z - \frac{i}{3}\right)} dz$$

Applying Cauchy's integral theorem

$$\frac{1}{3} \left[2\pi i f\left(\frac{i}{3}\right) \right] = \frac{1}{3} \left\{ 2\pi i \left[\left(\frac{i}{3}\right)^3 - 6 \right] \right\}$$

$$= \frac{1}{3} \left\{ 2\pi i \left[\left(\frac{i}{3}\right)^3 - 6 \right] \right\} = \frac{2\pi}{81} i^4 - 4\pi i$$

$$= \frac{2\pi}{81} - 4\pi i$$

32. Ans.(d)

Exp. Energy supplied by the motor

$$E_2 = 7 \text{ kW}$$

$$= 7000 \omega$$

$$= 7000 \text{ N.m / sec}$$

But energy absorbed during one riveting operation which takes 1 sec

$$E_1 = 10000 \text{ N.m}$$

Number of rivets that can be closed per minute

$$= \frac{E_2}{E_1} \times 60$$

$$= \frac{7000 \times 60}{10000}$$

$$= 42 \text{ Rivets}$$

33. Ans.(b)

Exp. Required probability = $p_7 + p_8$ where

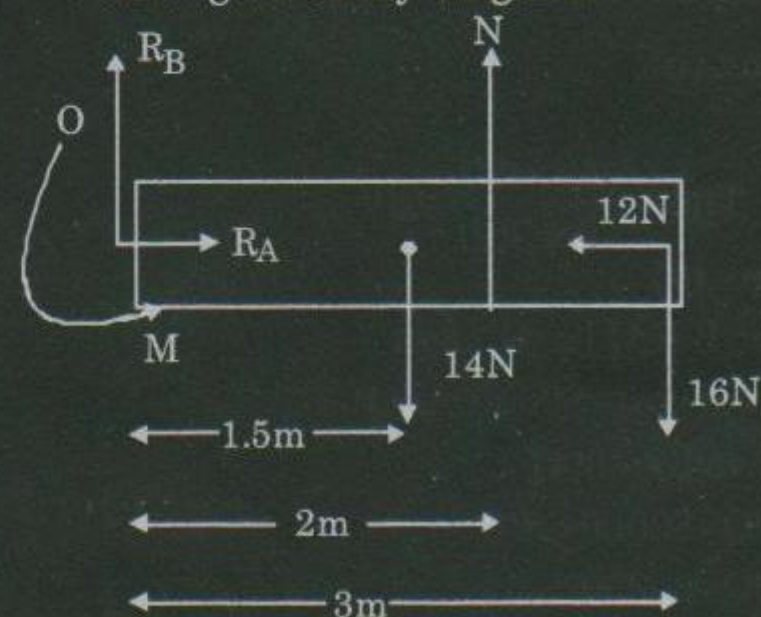
$$p_7 = \frac{1}{2} \times \frac{1}{11} + \frac{1}{2} \times \frac{6}{36} = \frac{1}{2} \left(\frac{1}{11} + \frac{1}{6} \right)$$

$$p_8 = \frac{1}{2} \times \frac{1}{11} + \frac{1}{2} \times \frac{5}{36} = \frac{1}{2} \left(\frac{1}{11} + \frac{5}{36} \right)$$

$$p_7 + p_8 = 0.244$$

34. Ans.(d)

Exp. Drawing free body diagram



$$R_A = 12\text{N}$$

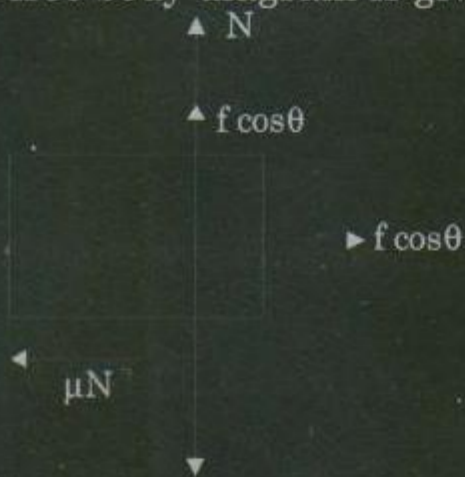
$$R_B + N = 16 + 14 = 30\text{N}$$

$$\text{Moment about O, we have } M + n \times 3 + 14 \times 1.5 = 48 + 21$$

4 Unknowns but only 3 equations

35. Ans.(d)

Exp. The free body diagram is given as



$$N = 10 - f \sin \theta$$

When it leaves surfaces

$$N = 0 = 10 - 4 t_1 \sin \theta$$

$$= t_1 = \frac{2}{\sin \theta}$$

For sliding $\mu N = f \cos \theta = 8 - 4 t_2 \sin \theta = 5 t_2 \cos \theta$

$$t_2 > t_1 = \frac{8}{4 \sin \theta + 5 \cos \theta} > \frac{2}{\sin \theta}$$

$$\Rightarrow \cos \theta < 0$$

$$\Rightarrow \theta_{\min} = 90^\circ$$

36. Ans.(b)

$$\text{Exp. } \omega = \frac{2\pi \times 1500}{60} = 157.08 \text{ rad/sec}$$

$$TR = \sqrt{\frac{1 + (2\xi r)^2}{(1 - r^2)^2 + (2\xi r)^2}}$$

$$0.15 = \sqrt{\frac{1 + (2 \times 0.08 r)^2}{(1 - r^2)^2 + (2 \times 0.08 r)^2}}$$

$$r = 2.816$$

$$r = \frac{\omega}{\omega_n}$$

$$\omega_n = \frac{\omega}{r} = \frac{157.08}{2.816} = 55.78 \text{ rad/sec}$$

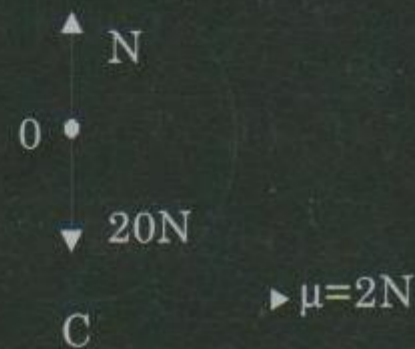
$$\therefore \omega_n = \sqrt{\frac{k}{m}}$$

$$\therefore k = 650 \times 55.78^2$$

$$K = 2.02 \text{ MN/m}$$

37. Ans.(c)

Exp. The free body diagram is given as



$$a = \frac{2N}{2} = 1 \text{ m/s}^2$$

$$\mu = \frac{1}{0.1} = 10 \text{ rad/s}^2$$

The frictional force becomes non-dissipative and negligible when instantaneous velocity of c becomes zero.

$$\text{i.e. } v - \omega r = 0$$

$$\Rightarrow at - (\omega_0 - \mu t)r = 0$$

$$\text{So } 10 - 20t = 0$$

$$\Rightarrow t = 0.5 \text{ sec}$$

38. Ans.(b)

39. Ans.(c)

Exp. $C > 2\sqrt{km} = 2\sqrt{500} \Rightarrow 50$ over damped system

$$s_1, s_2 = -\frac{100}{2 \times 5} \pm \left(\frac{100^2}{4 \times 25} - \frac{100}{5} \right)^{1/2}$$

$$= -18.94, -1.06$$

$$\text{So } x = A e^{-18.94t} + B e^{-1.06t}$$

40. Ans.(a)

41. Ans.(b)

42. Ans.(a)

$$\text{Exp. } T_e = T_i \left(\frac{P_e}{P_i} \right)^{\frac{\gamma-1}{\gamma}}$$

$$T_e = 350 \left(\frac{1}{4} \right)^{1.4}$$

$$T_e = 235.5 \text{ k}$$

$$\therefore C_p = \frac{\gamma R}{\gamma - 1}$$

$$\therefore C_p = \frac{8.314 \times 1.4}{0.4}$$

$$C_p = 29.1 \text{ J/mol - k}$$

$$\bar{C}_P = \frac{29.1}{29 \times 10^{-3}} = 1.0034 \text{ KJ / kg - k}$$

$$V_e = \sqrt{2C_P (T_i - T_e)}$$

$$V_e = \sqrt{2 \times 1.0034 (350 - 235.5) \times 10^3}$$

$$\boxed{V_e = 479.35 \text{ m / s}}$$

43. Ans.(d)

44. Ans. (a)

Exp. Discharge passing between the stream lines is given by

$$\psi_1 - \psi_2$$

$$\psi_1 (1, 3) = 6$$

$$\psi_2 (5, 5) = 0$$

$$\therefore \psi_1 - \psi_2 = 6 \text{ unit}$$

45. Ans. (b)

Exp. $u_1 = 200 \text{ KJ / kg}$ (given)

$$U_2 = mu_1 \Rightarrow U_1 = .25 \times 200$$

$$U_1 = 50 \text{ KJ}$$

$$u_2 = 300 \text{ KJ / kg}$$
 (given)

$$U_2 = mu_2 \Rightarrow U_2 = .75 \times 300$$

$$U_2 = 225 \text{ KJ}$$

$$\text{Total } U = U_1 + U_2$$

$$U = 275 \text{ KJ} \quad U = u \times m$$

$$u = \frac{U}{m} \quad u = \frac{275}{1}$$

$$\boxed{u = 275 \text{ KJ / kg}}$$

46. Ans. (d)

Exp. Shear stress (f_s) = $\frac{P}{2b.t}$

$$f_s = \frac{60 \times 10^3}{2 \times 40 \times 15}$$

$$f_s = \frac{60 \times 10^3}{2 \times 600} \quad f_s = \frac{1}{2} \times 10^3$$

$$\boxed{f_s = 50 \text{ N / mm}^2}$$

47. Ans. (a)

$$\begin{aligned} \text{Exp. } \omega_D &= (2V_1 \cos \alpha - V_b) V_b \\ &= [2 \times 700 \times \frac{1}{2} - 300] \times 300 \\ &= 120 \text{ kw} \end{aligned}$$

48. Ans.(d)

Exp. Maximum shear force = vertical reaction at A

$$V_A = \frac{1}{3} \times 8 \times 5 = 13.33 \text{ KN}$$

49. Ans.(a)

Exp. Maximum SF occurs at the level of neutral axis at the section of maximum SF. for square sections.

$$\tau_{\max} = \frac{3}{2} \times \tau_{\text{avg}} = \frac{3}{2} \times \frac{13.33 \times 10^3}{(40 \times 40)} = 12.5 \text{ N / mm}^2$$

50. Ans. (c)

$$\text{Exp. } u = -\frac{\partial \psi}{\partial y} = -4x$$

$$v = \frac{\partial \psi}{\partial x} = 4y$$

For point B,

$$x = 3 \text{ m}$$

$$y = 1 \text{ m}$$

$$u = -0.12 \text{ cm/sec}$$

$$v = 0.04 \text{ cm/sec}$$

$$v = \sqrt{u^2 + v^2} = 0.126 \text{ cm / sec}^2$$

51. Ans.(d)

Exp. The convective acceleration

$$a_x = u \frac{\partial u}{\partial x} + v \frac{\partial v}{\partial y}$$

$$a_y = u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y}$$

$$\frac{\partial u}{\partial x} = -4, \frac{\partial u}{\partial y} = 0 \text{ and } \frac{\partial v}{\partial x} = 0, \frac{\partial v}{\partial y} = 4$$

$$a_x = -0.12(-4) \times 10^{-4} = 48 \times 10^{-6} \text{ cm / s}^2$$

$$a_y = 0.04 \times 4 \times 10^{-4} = 16 \times 10^{-6} \text{ cm / s}^2$$

$$a = \sqrt{a_x^2 + a_y^2} = 50.6 \times 10^{-6} \text{ cm / s}^2$$

52. Ans.(d)

Exp. By interpolation degree of super heat

$$\bar{C}_P = \frac{29.1}{29 \times 10^{-3}} = 1.0034 \text{ KJ / kg - k}$$

$$V_e = \sqrt{2C_P (T_i - T_e)}$$

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